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# Parallel Solution of the Wave Equation Using Higher Order Finite Elements

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*This paper is dedicated to Roland Glowinski on the occasion of his 60th birthday*

## 1. Introduction

The time domain simulation of wave propagation phenomena is a computationally demanding task. The acoustic wave equation is the simplest such model and serves as a useful benchmark for more realistic situations (elastodynamics, or electromagnetism). This paper presents a parallel simulation code for such phenomena. The initial implementation is for 2D acoustics, but of course the method is general, and we are currently investigating more complex models.

We use the higher order finite elements developed by Cohen et al. in [CJT95]. These elements were designed to give a diagonal mass matrix, thus enabling an explicit solution, while retaining high accuracy. They are based on a modification of the classical  $P_2$  and  $P_3$  elements, and are described briefly in section 2.1. We also recall how the modified equation technique leads to higher order methods in time. As the resulting method is explicit, it lends itself very naturally to a parallel implementation. We have chosen a coarse grain, domain splitting approach, using message passing, as this is known to be the most portable approach, likely to give the best efficiency on a wide range of parallel computers. Our implementation is detailed in section 3., and we present numerical results in section 4..

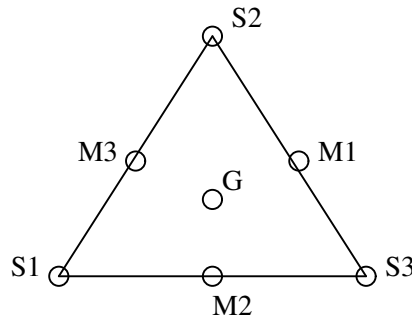
## 2. Discretization of the wave equation

### 2.1. Finite element spaces with mass lumping

When solving wave propagation problems with finite differences, it is usual to employ explicit methods because the time steps are dictated more by accuracy than by stability constraints. One wants to do the same when solving with finite elements. However, except for the lowest order (P1) elements, the mass matrix will not be diagonal, and one would still have to solve a linear system at each time step. This makes the method unacceptably costly, and has led Cohen et al. [CJT95] to design a new family of finite elements, with the goal of achieving high accuracy, while keeping the mass matrix diagonal. This is done by modifying the classical finite element space (adding new degrees of freedom), so that quadrature formulas with positive weights can be found. Positive weights are essential for preserving the stability of the scheme. We describe the new elements, referring the reader to [CJT95, Tor95, CJT96] for the details of the construction.

Making the mass matrix diagonal simply means that the nodes of the quadrature formula have to be the degrees of freedom of the element. In order to retain sufficient accuracy, the quadrature formula must be exact for some polynomial space, whose exact degree has can be found in the above reference (the result is due to Ciarlet). Last, as we said above, the weights of the quadrature formula must be positive. It turns out that these constraints cannot all be met simultaneously for the classical  $P_2$  and  $P_3$  spaces (a unique quadrature formula with the right degree can be found in each case, but have either zero, or negative weights).

For  $P_2$ , the new polynomial space is  $\tilde{P}_2 = P_2 \oplus b$ , where  $b = \lambda_1 \lambda_2 \lambda_3$  is a ‘‘bubble’’ function. The additional degree of freedom is the function value at the center of mass of the element. The quadrature formula is simply Simpson’s rule. We show the degrees of freedom of the element on figure 1



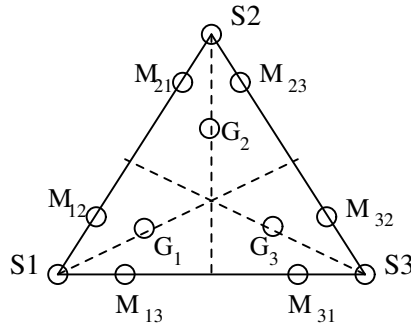
**Figure 1** Degrees of freedom of the  $\tilde{P}_2$  element

For  $P_3$ , the degrees of freedom are the values of the function at:

- the three vertices;
- two points on each side, at a distance  $\alpha$  from the vertex,
- three interior points, with barycentric coordinates  $(\beta, (1 - \beta)/2, (1 - \beta)/2)$  (up to

a circular permutation).

( $\alpha$  and  $\beta$  are parameters whose values can be found in the above references). The polynomial space is  $\tilde{P}_3 = P_3 + bP_1$  (of dimension 12). The basis functions at the three interior points are bubble type functions. It can be checked that the set of nodes is  $\tilde{P}_3$  unisolvent, and that a (unique) quadrature formula with positive weights can be found, which uniquely determines  $\alpha$  and  $\beta$ . The degrees of freedom for this element are shown on figure 2



**Figure 2** Degrees of freedom of the  $\tilde{P}_3$  element

It is proved in [Tor95] that one gets error estimates proportional to  $h^2$  for  $P1$  elements, and to  $h^4$  (resp.  $h^6$ ) for  $\tilde{P}_2$  (resp.  $\tilde{P}_3$ ) elements.

2.2. Time discretization

The finite element spaces described above are coupled to a time stepping scheme to obtain a fully discrete solution. We first use the classical, second order accurate, leap-frog scheme:

$$M \frac{u^{n+1} - 2u^n + u^{n-1}}{\Delta t^2} + K u^n = f^n \tag{1}$$

where  $M$  is the mass matrix (which is diagonal, thank to the choices we made above), and  $K$  is the stiffness matrix. This scheme is centered, explicit, second-order accurate in time, and conditionally stable, with a CFL condition  $c\Delta t/h \leq \alpha_2$ . The value of  $\alpha_2$  is given in [CJT95].

This has the drawback that the the accuracy order in the *space* variables is effectively reduced to 2. It is thus useful to employ a fourth order accurate scheme in time. In order to obtain an explicit scheme, we follow [CJ90] and derive the scheme via the modified equation technique. The scheme thus obtained is fourth order in time, still explicit, and subject to a CFL condition slightly more restrictive than for the second order scheme. For details, see [CJT95]. Note that the cost of the fourth order scheme is double that of the second order scheme, which will hopefully be offset by the larger time steps one may use.

### 3. A parallel implementation

#### 3.1. Algorithm

Based on these techniques, we have written a parallel simulation code for the 2D acoustic wave equation. Parallelism is based on a domain splitting approach (with non-overlapping domains), and keeps the explicit nature of the scheme. This corresponds to the fully explicit treatment given by Quarteroni [Qua95]. Each processor handles one subdomain. At every time step, all processors update their interior nodes (in parallel), then exchange interface nodes with their neighbors. In contrast to domain decomposition methods for elliptic problems, our procedure is algebraically equivalent to the sequential one.

A different strategy (introduced in [MS87]) would be to use overlapping subdomains, and to take advantage of the hyperbolic nature of the problem by introducing a partition of unity, thus decomposing the initial condition in 2 functions with disjoint support. Because of the finite speed of propagation, the solutions in the 2 subdomains remain disjoint for some time. Thus communication occurs infrequently. A major drawback of this scheme is that overlapping will entail a major memory penalty, especially for 3D problems. This has deterred us from using this method.

Still a different method is used by Dean and Glowinski in [DG94]. They explicitly enforce continuity of the function across interfaces by using a Lagrange multiplier. They have then to solve a linear system on the interface at each time step, but this system is small. The advantage of their method is that it handles non-matching meshes, something we are unable to do.

We shall detail the “matrix vector” product, as this is where almost all of the parallelism is hidden. We must emphasize that we do not use and distribute a global matrix. Each processor has a full subdomain, and interface nodes are duplicated between neighboring processors. For simplicity, let us assume we are dealing with two subdomains. We start from the obvious remark that:

$$\int_{\Omega} (\nabla u_h^m \cdot \nabla w^i)(x) dx = \int_{\Omega_1} (\nabla u_h^m \cdot \nabla w^i)(x) dx + \int_{\Omega_2} (\nabla u_h^m \cdot \nabla w^i)(x) dx. \quad (2)$$

where  $u_h^m$  is the discrete solution at the  $m$ th time step, and  $w^i$  is a basis function. Let us number first the nodes in  $\Omega_1$ , then those in  $\Omega_2$ , and last the nodes on the interface, and partition the stiffness matrix  $K$  accordingly. Then, with obvious notation,

$$K = \begin{pmatrix} K_{11} & 0 & K_{13} \\ 0 & 0 & 0 \\ K_{31} & 0 & K_{33}^1 \end{pmatrix} + \begin{pmatrix} 0 & 0 & 0 \\ 0 & K_{12} & K_{23} \\ 0 & K_{32} & K_{33}^2 \end{pmatrix} \quad (3)$$

and we perform the matrix vector product  $Kv$  as follows [LP96]:

1. Compute in processor  $i, i = 1, 2$ :

$$v^i = \begin{pmatrix} K_{ii} & K_{i3} \end{pmatrix} \begin{pmatrix} u^i \\ u^3 \end{pmatrix} \quad v_i^3 = K_{3i}u^i \text{ and } v_{3i}^3 = K_{33}^i u^3. \quad (4)$$

2. Send  $v_1^3$  and  $v_{31}^3$  (resp.  $v_2^3$  and  $v_{32}^3$ ) to processor 2 (resp. 1).

3. In both processors, let  $v^3 = v_1^3 + v_2^3 + v_{31}^3 + v_{32}^3$ .

This is easily implemented, assuming each processor has

- local information (solution, mass and stiffness matrices),
- a list of neighboring processors and interface nodes.

The structure of the code looks like:

```

for each neighbor
  post receive request (goes to a buffer)
do local matrix vector product
for each neighbor
  send interface nodes
wait on all receive requests
for each neighbor
  add buffer to proper interface node location

```

### 3.2. MPI evaluation

Our strategy leads naturally to a message passing implementation. We have chosen to use MPI [For94] in order to evaluate its ease of use and expressiveness, and also to benefit from its portability.

Contrary to our initial fears, we found MPI rather easy to learn (at least as far as basic features are concerned). The book by Gropp et al. [GLS94] was most useful.

In the finite element context, the most useful feature are the derived data-types. The indices of the nodes to be exchanged at each time steps are spread over the whole index set. Thus a indexed data-type is the natural way to access these elements. Unfortunately, this only works on the sender side. On the receiver side, we had to use a separate buffer, and add the elements by hand to their proper location. This is because MPI does not have a scatter-add function, an observation that has already been made (e.g. by Smith [SM95]).

Since this was easy to implement, we also used non-blocking receives: at the beginning of each time step, each node posts its receives, updates all of its node from local information, then sends the interface nodes to its neighbors, and waits on its receives. We were not able to quantitatively evaluate the usefulness of this approach, but we felt it could provide an advantage over simple “compute, send, receive” using blocking receives, and it was just as easy to program.

One feature we did not use, but which could probably make the code somewhat simpler (and maybe give some more efficiency) are persistent requests. Since we keep a fixed mesh, the nodes to be exchanged at each time step are the same. It would thus be possible to predefine all the send and receive requests, and simply start them (and wait for them) at each time step. It has for the moment proven difficult to integrate this feature into the code. We need to provide the send and receive buffers we shall actually use in the communication at the time the requests are defined, and this conflicts with the modular way the code is structured.

Finally, the results we report below were obtained with several different MPI implementations: the ANL-MSU MPICH implementation [DGLS93] (on both our

local workstations and the Cray T3D), the IBM MPI-F [FWR<sup>+</sup>95] and the SGI implementation. The only portability problem we had was that we used the MPE extensions for logging (see section 4.2., and these are not part of the MPI standard, they are only provided by MPICH. Apart from this (obvious) point, no changes were necessary. It is quite remarkable how quickly after the standard appeared vendors were able to provide high quality implementations.

## 4. Numerical results

### 4.1. Code development, pre and post processing

The code has been developed and tested on a network of Digital Alpha workstations (this is yet another nice feature of MPI), and then ported to a Cray T3D, an IBM SP/2 and an SGI Power Challenge Array.

We used pre- and post-processing tools from the Modulef group. Modulef is an extensive finite element library( see [BGJ<sup>+</sup>86]), but we used only the mesh generation, mesh decomposer and visualization modules. We comment briefly on the last two, as they were key points in the project. The `decomp` mesh decomposer [TSV94] is somewhat unusual in that it is based on clustering techniques from data analysis, and not on a spectral or a greedy technique. Though we have no comparison with other methods, this technique has proved to give balanced sub-domains in several applications. It is also worth noting that figure 3 was produced with the `visu` program, also from the Modulef group. This visualization program lets us work with several meshes and their associated solutions at the same time, an essential feature in our context.

### 4.2. Performance

In this section we give some performance data on 2 different parallel computers. The model used is the homogeneous unit square, excited by a Ricker wavelet (a quasi point source, with a time dependence the second derivative of a Gaussian function). The experiment is the equivalent of dropping a stone at the center of a square pond.

We ran the code on a T3D at CEA-CENG, and on an IBM SP/2 at CNUSC. We used the same model problem as in the previous section, with 2 different mesh resolution: the coarse mesh is a  $49 \times 49$  subdivision of the unit square, leading to 30241 degrees of freedom (we use  $\tilde{P}_3$  elements throughout). This is a relatively small problem. The fine mesh is twice as large: a  $99 \times 99$  subdivision of the unit square, leading to 125441 degrees of freedom. This is still not a large problem.

We also have two versions of the code, and could unfortunately not run all of the experiments with the same version. Version 1 was an attempt at using an unassembled stiffness matrix, thus using very little memory. This was found to use too much time, and we switched to an assembled stiffness matrix. This is referred to as version 2. Basically, all IBM results pertain to version 1, and all Cray results pertain to version 2. This has the unfortunate effect of making comparison between machines impossible, but this was not the goal of this paper anyway. Several good benchmarks have been published to this effect. Let us just mention that in the few cases where we ran the

Nb CPUS	1	2	4	8	16
Time (s)	1.37	0.7	0.37	0.25	
Speedup	1	1.96	3.70	5.48	
Efficiency	1	0.98	0.93	0.69	
Average # DOFs	30200	15200	7800	3800	
Time (s)	5.57	2.8	1.43	0.7	0.44
Speedup	1	1.99	3.90	7.95	12.8
Efficiency	1	0.99	0.98	0.99	0.80
Average # DOFs	125000	62800	31900	15800	7900

**Table 1** Speedup and efficiency on a SP/2,

same code on both machines, the SP/2 appeared to be about 60% faster than the T3D.

We show on table 1 results for version 1 of the code, on the IBM SP/2 (first for the coarse mesh, then for the fine mesh). The times are time per time step (since the mesh is refined for the second part of the table, the time step was also divided by 2).

We also show on table 2 the corresponding measurements on the T3D (for version 2 of the code, with again coarse mesh first, then fine mesh, and times are per time step):

Nb CPUS	1	2	4	8	16
Time (s)	0.44	0.24	0.14	0.08	
Speedup	1	1.86	3.23	5.68	
Efficiency	1	0.93	0.81	0.71	
Time (s)	0.93	0.51	0.27	0.14	0.08
Speedup	1	1.83	3.50	6.72	12.2
Efficiency	1	0.91	0.88	0.84	0.77

**Table 2** Speedup and efficiency on a T3D

These results call for several comments.

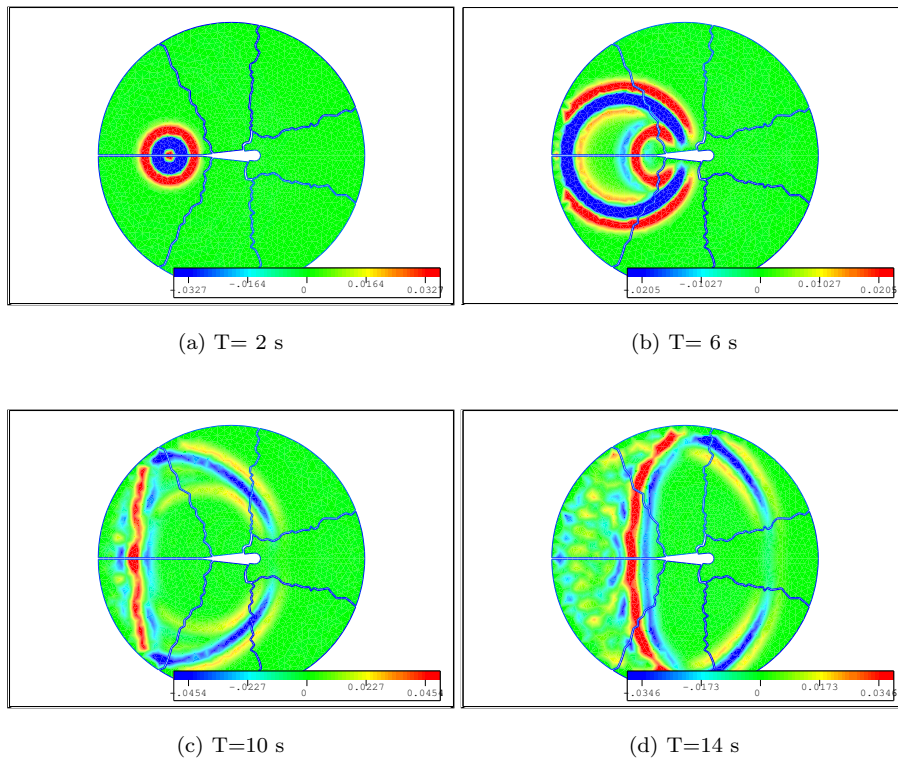
- For version one of the code, one finds that the times for the fine mesh are about 4 times as large as for the coarse mesh. This makes sense, since we have 4 times as many elements, and the stiffness matrix is unassembled. This is no longer true for version 2, where a time step takes about twice as long, even though the matrix has 4 times as many nonzeros for the fine mesh. It is not quite clear why this is so.
- As far as scalability is concerned, the observed results agree with the usual expectations: for each problem size, there will be a maximum number of processors that can be profitably utilized. For the smaller problem size, this number is somewhere between 4 and 8, and for the larger size it is between 8 and 16.
- It seems that when one keeps the number of degrees of freedom per processor



constant, the time per time step decreases with increasing number of processors

#### 4.3. Diffraction by an obstacle

We have also looked at a more interesting example, namely the diffraction of a point source by an obstacle. We have chosen to redo an example from N. Tordjman's thesis, where the obstacle is a cone. To take full advantage of finite elements, we use a circular artificial boundary and a completely unstructured mesh, split into 7 subdomains by the `decomp` mesh decomposer. On figure 3, we display several snapshots, showing how the wavefronts moves as time increases.



**Figure 3** Snapshots for the cone example

As one can see from the figures, we have used simple Dirichlet boundary conditions on the artificial boundary, so as to obtain a more spectacular wavefront. It is also apparent that the front suffers no discontinuity at the interface between two subdomains. Figure 3(a) should be compared with figure 12 in [CJT96], and this shows good qualitative agreement.

This example was computed on an SGI Power Challenge Array, and took 27s on 7 processors, compared to 96s on one, for a speedup of 3.55. This problem is clearly too

small for this number of processors. The average number of degrees of freedom per subdomain is 3900.

## 5. Conclusions

We have implemented a parallel solver for the acoustic wave equation, using the MPI standard. This code runs on several massively parallel computers. For small models, relatively good efficiency can be attained on a few processors.

There remains to run larger models, in order to attempt a scalability analysis. However, we believe that realistic models for the 2D wave equations will not be large enough for this task (from another point of view, the sizes needed for this study will be much too large for what is usually done in acoustics modeling).

We are currently investigating 3D finite elements on tetrahedra, and have already obtained theoretical results with a second order element.

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